

Thermal management architectures for rack based electronics systems

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#### Abstract

Cooling architecture options are studied for rack based electronics such as computer servers or router line cards. For the purposes of this investigation, individual servers or line cards are assumed to be of the horizontal pizza box style with a few high power modules such as CPU's or high power ASICS and a number of other lower power packages, memory, I/O chips etc. A number of system architectures for cooling the high power modules are considered and the corresponding performance levels that can be achieved are evaluated. In the first, fairly conventional cooling architecture, a "keep in volume" is allocated around each high power module for local air cooled heat sinks. In the second option, volume for air cooled fin surfaces is provided above the card footprint within the server volume but remote from the high power modules such that a single or two-phase heat transport means is required to transport heat from the modules to the fins in the remote location. In a third option, the heat transport loop at the card level transfers heat from the high power modules to a rack level liquid cooling system that dissipates heat to air flowing through the rack. In a fourth architecture, the heat transport loop at the card level connects to a rack level liquid cooling system that exchanges heat with circulating liquid supplied by a cooling system external to the rack. These options are selected to provide a broad look at cooling design possibilities and tradeoffs. Cooling architecture choices are more limited within racks that contain a variety of servers made by different vendors. The more powerful architectures are feasible within racks where a single vendor owns all the functions and has complete flexibility to consider rack level design tradeoffs.



## Outline

- Typical products of interest
- Layout & design specs considered
- Architecture options
  - Constraints
  - Conceptual models
- Thermal Assessment
  - Methodology
  - Performance metrics & comparisons
- Wrap up



## Typical products of interest



#### Servers ...

#### Router line cards ...



# Simplified layout considered





## Simplified layout considered



# **Cooling Architectures**

- 1. Air cooled heat sinks are located on heat sources within "keep in volumes" allocated around each high power module
- 2. A single or two-phase heat transport means is provided to transport heat from the high power modules to a <u>remote air cooled heat</u> <u>exchanger located above the card footprint</u> within the server volume.
- 3. A single or two-phase heat transport loop at the card level transfers heat from the high power modules <u>through a conduction connection</u> to a rack level liquid cooling system. The rack level liquid cooling system can dissipate heat to air flowing through the rack or to chilled water supplied by a room level utility.
- 4. A single phase heat transport loop at the card level connects <u>through liquid couplings to a rack level liquid cooling system</u>. The rack level liquid cooling system can dissipate heat to air flowing through the rack or to chilled water supplied by a room level utility.



#### 1: Heat sink mounted directly on CPU

Air cooled finned heat sink with heat pipe or vapor chamber enhanced base situated within "keep in volume" around module.



#### 2a: Passive two phase transport w/air cooled HEX on Card

Heat pipe, thermosiphon or loop heat pipe solution with air cooled condenser located within the server enclosure.



# 2b: Pumped single phase transport with HEX on Card

Liquid cooling loop with pump and liquid-to-air heat exchanger within the server enclosure.



# 3: On Card loop with thermal connection to Rack level cooling system

Passive or pumped liquid loop with heat exchange by conduction connection to rack level liquid cooling system.



# 4: Rack level liquid flows directly to card level cooling system

Rack level pumped liquid flows directly to cold plates mounted on CPU or ASIC modules.



#### 3 & 4: Rack level liquid cooling system





## **Cooling Assessment Overview**

For any heat exchanger [1]

$$Q = \left(\dot{m}c_p\right)_{min} \cdot \left(\Delta T_{app}\right) \cdot \varepsilon$$

Simplified version for heat sinks

$$Q = \dot{m}c_p \cdot (T_s - T_i) \cdot (1 - e^{\frac{m_s n_s}{\dot{m}c_p}})$$

 $-h\Delta n$ 

Pressure drop

$$\Delta p = \frac{1}{2} \rho \cdot U_m^2 (K_c + K_e + f \cdot \frac{L}{D_h})$$



# **Methodology Simplifications**

For Heat Exchangers with two fluid streams:

- Air and water "h" values are calculated using appropriate correlations [2]
- Condensation "h" set at ~5 kW/m<sup>2</sup>-K

Evaporation "h" set at ~60kW/m<sup>2</sup>-K



#### **Design specs considered**

- Input Power per CPU:
- CPU Heat Footprint:
- Air flow per CPU:
- Air Inlet Temp:
- Liquid Flow per CPU:
- Liquid type:
- Chilled Water Temp:

250 Watts 25x25 mm2

50 CFM 35 °C

0.75 LPM water

18 °C



#### **Design specs considered**

Local Heat Sink Size: 103Wx75Lx28H

• On Board HEX Size: 105Wx30Lx35H (Up to 300 mm forward of heat sources)

• Rack side HEX Size: (At air inlet side of rack) 105Wx30Lx44H



## **Performance Comparison**

	1		2a	2b	3a	3b	4a	4b
Thermal Architecture	Local Heat Sinks on high power modules		Heat transport loop with remote condenser in server		Server two-phase cooling loop with conduction connection to rack cooling system		Server liquid loop with liquid couplings to rack liquid cooling system	
Version	4 module	es 2 modules	two phase passive loop	single phase pumped loop	rack liquid to rack air HEX	rack liquid to water HEX	rack liquid to rack air HEX	rack liquid to water HEX
Та	35	35	35	35	35	18	35	18
Design Constraint								
HX width	105	210	105	105	105		105	
HX length	75	75	30	30	30		30	
HX height	28	28	35	35	44		44	
Resistances					S MARINE MARINE		HHHH	
Rfa			0.127	0.127	0.084	0.054	0.084	0.054
Rff					0.005	0.005		
Rsf	0.2	0.16	0.033	0.023	0.033	0.033	0.023	0.023
Rtim	0.02	0.02	0.02	0.02	0.02	0.02	0.02	0.02
Total Rc-a	0.22	0.18	0.18	0.17	0.142	0.112	0.127	0.097
Pressure Drops								
DP air (Pa)	100	50	100	100	100		100	
DP liq				15000	30000	30000	30000	30000
Performance Summa	iry							-
Case Temp (°C)	90	80	80	77.5	70.5	46	66.8	42.3
Fan Power (W)	9.4	4.7	9.4	9.4	9.4	0.0	9.4	0.0
Pump Power (W)	0.0	0.0	0.0	0.4	0.8	0.8	0.8	0.8



## **Selection Metrics**

	1		<b>2</b> a	2b	<b>3</b> a	3b	4a	4b
Thermal Architecture	Local Heat Sinks on high power modules		Heat transport loop with remote condenser in server		Server two-phase cooling loop with conduction connection to rack cooling system		Server liquid loop with liquid couplings to rack liquid cooling system	
Version	4 modules	2 modules	passive	pumped	rack air HEX	rack water HEX	rack air HEX	rack water HEX
Case Temperature (°C)	90	80	80	77.5	70.5	46	66.8	42.3
Pumping Power (W)	10	5	10	10	11	1	11	1
Mass (kg)	0.4	0.8	0.36	0.42	0.46	0.4	0.46	0.4
Mass on CPU (kg)	0.4	0.8	0.2	0.2	0.2	0.2	0.2	0.2
Cost per CPU	0.5X	1X	1.1X	1.4X	2X	1.8X	2X	1.8X
Failure Modes	Micro- leaks, Freeze- Thaw	Micro- leaks, Freeze- Thaw	Micro- leaks	Pumps	Pumps, Thermal Connect	Pumps, Thermal Connect	Pumps, Liquid Coupling	Pumps, Liquid Coupling
L2 Life (years)	10	10	7	5	5	5	4	4



# **Concluding Remarks**

- A number of cooling architectures are evaluated for rack based electronics systems. Cooling performance as well as cost and reliability estimates are made.
- Options 1 and 2 can be implemented even if the rack contains cards from multiple independent suppliers.
- Option 3 and 4 can be used when the rack level design is well integrated with the card level solutions, usually from a single supplier.
- Options 3 and 4 are most powerful when the rack level solution includes heat exchange with the end users facility level chilled water.



#### References

- 1. Kays, W.M. and London, A.L., Compact Heat Exchangers, 3<sup>rd</sup> Ed., McGraw Hill, NY, 1984
- 2. Handbook of Single-Phase Convective Heat Transfer, S. Kakac, R.K. Shah and W. Aung, eds., Wiley, New York, 1987





## Nomenclature

- heat transfer surface area As
- specific heat
- C<sub>p</sub> D<sub>h</sub> hydraulic diameter
- friction factor
- inertial pressure loss factor Κ
- length of flow path L
- heat transfer coefficient h
- mass flow rate m
- pressure р
- Q heat transfer rate
- T<sub>i</sub> inlet temperature
- T<sub>s</sub> surface temperature
- velocity through minimum flow cross sectional area

#### **Greek Symbols**

- density ρ
- heat transfer surface efficiency η<sub>s</sub>





#### **Illustrative Pictures**



#### 2A: LHP passive two-phase, HX in server volume





#### 2B: Pumped liquid cooling, HX in server volume





#### 3: Server loop conduction connected to rack loop





#### 4: Card level loop with liquid couplings to rack loop

Zero-Leak fluid couplings plug into liquid manifolds in rack



Fluid path on cold plate to manage non-uniform heat loads on card



#### 4: Water cooled rack



IBM Research's Water Cooled Chips: Scientists at IBM's Zurich Research Lab are working on the future of water cooling, bringing cold water to the hottest part, directly on the chip itself, and then capturing the water at its hottest and piping it off the chip for reuse. Shown here is an array of the chips in a rack, with the cold water going in, capturing the heat, and then the hot water is pumped out.

Source: http://picasaweb.google.com/ibmphoto/IBMSystems#5187345288745090450

